
Motor compartment size optimization for minimizing yaw motion in small overlap crash

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Motivation

Slide away motion



<https://news.hmgjournal.com/Tech/hyundai-sonata-platform-safety>

Yaw motion after crash



<https://www.youtube.com/watch?v=KNfZSgZ3p1k>

Effort to reduce yaw motion

Example : Hyundai 3rd platform

At the motor compartment

- Increase energy absorption
- Optimize load path
- **Higher strength of frames**



<https://news.hmgjournal.com/Tech/hyundai-sonata-platform-safety>

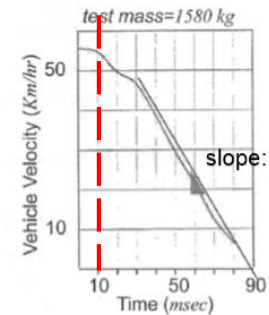
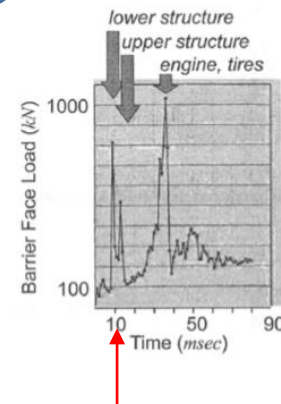
Concept and Modeling

Situation

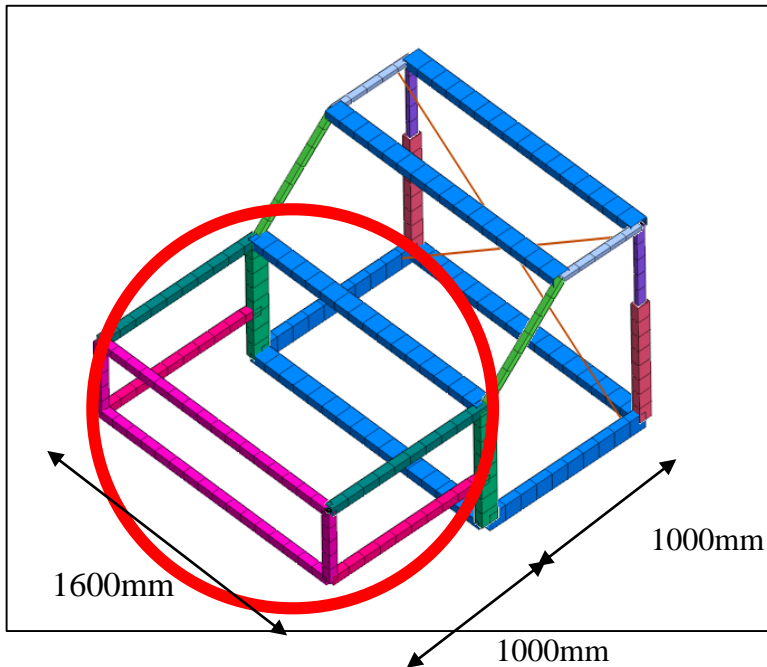
- Right after crash
- Before buckling of upper rail and mid rail
- Elastic deformation

Minimize yaw

- By changing strength of rails (length, thickness)

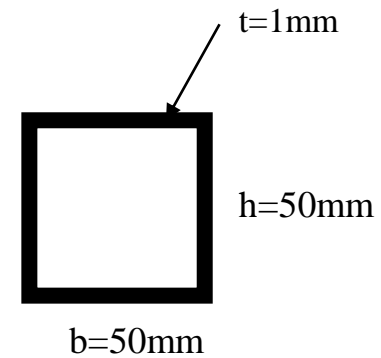


Geometry



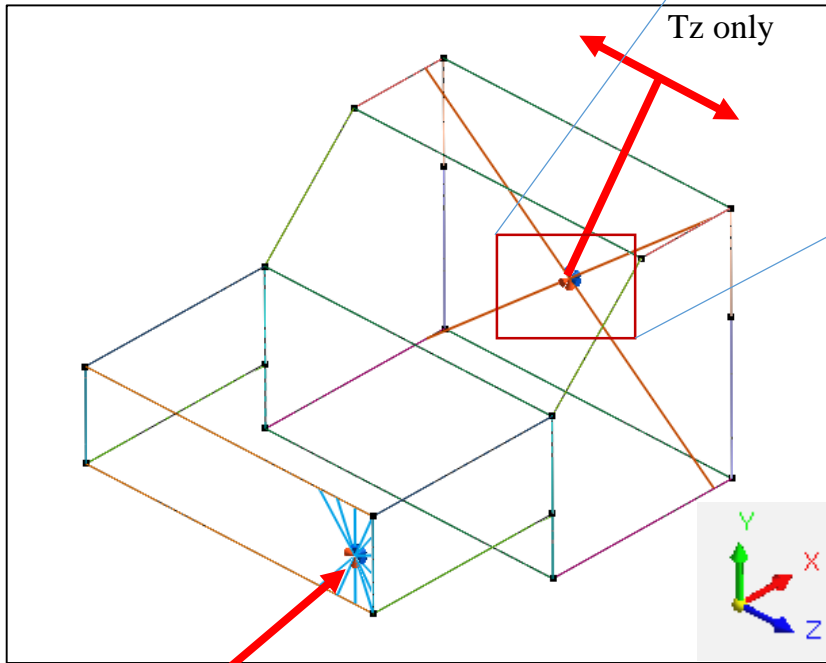
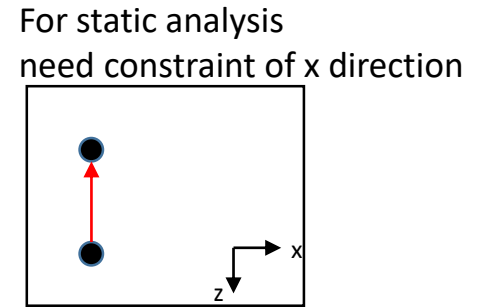
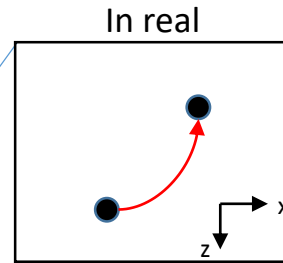
Motor compartment :
general assumption

Cross section and cabin size :
from lecture note

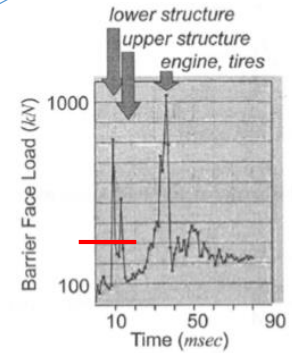


Simulation

Boundary condition

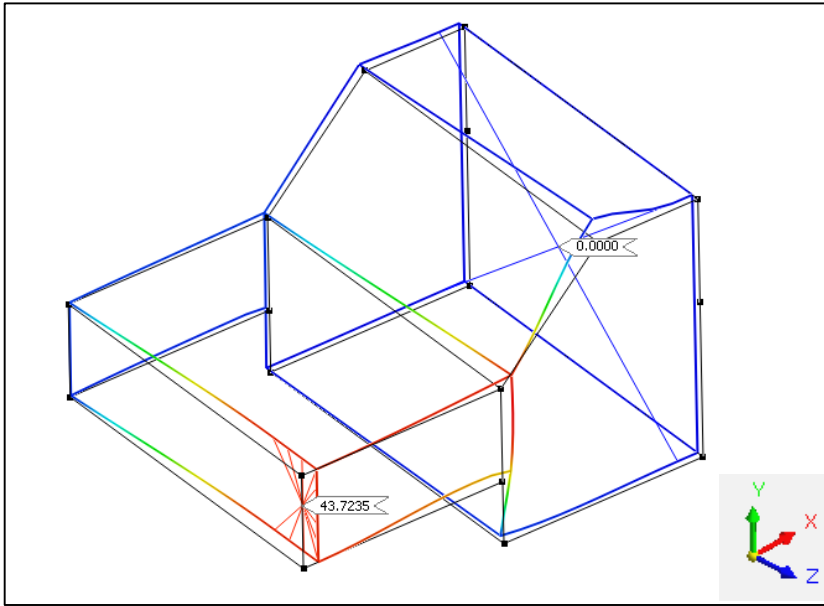


small overlap F
Tx only (barrier impact)

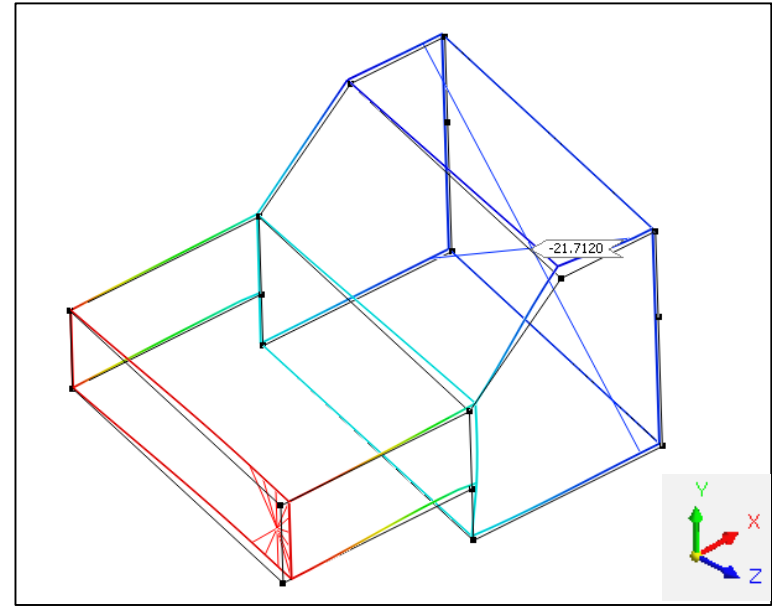


$F = 20g \times 1600kg$
 $F = 300kN$

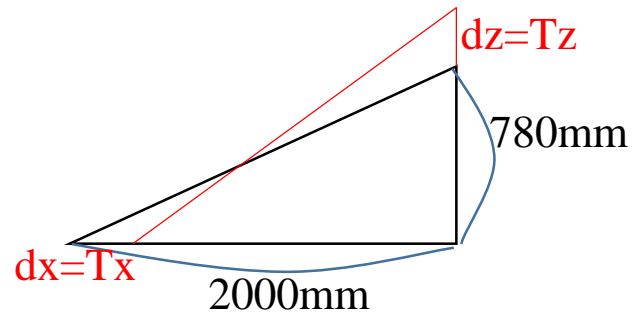
Simulation



$T_x = 43.7\text{mm}$



$T_z = 21.7\text{mm}$



$\Delta\theta = 0.98\text{deg}$

Size Optimization

Design variables

8 variables

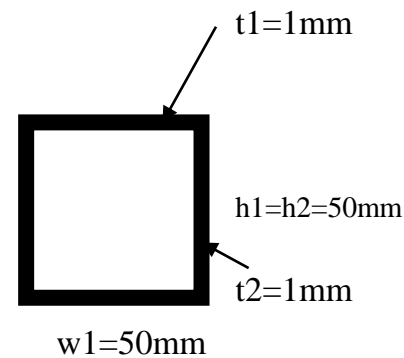
Width and height

- w_1, w_2, h_1, h_2
- w_1, w_2, h_1, h_2

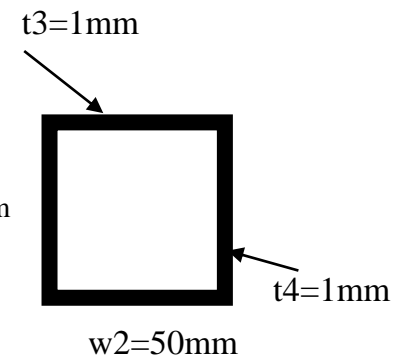
Thickness

- t_1, t_2, t_3, t_4
- t_1, t_2, t_3, t_4

Upper rail



Midrail



Objective function and Constraints

Objective function

Displacement of cabin center

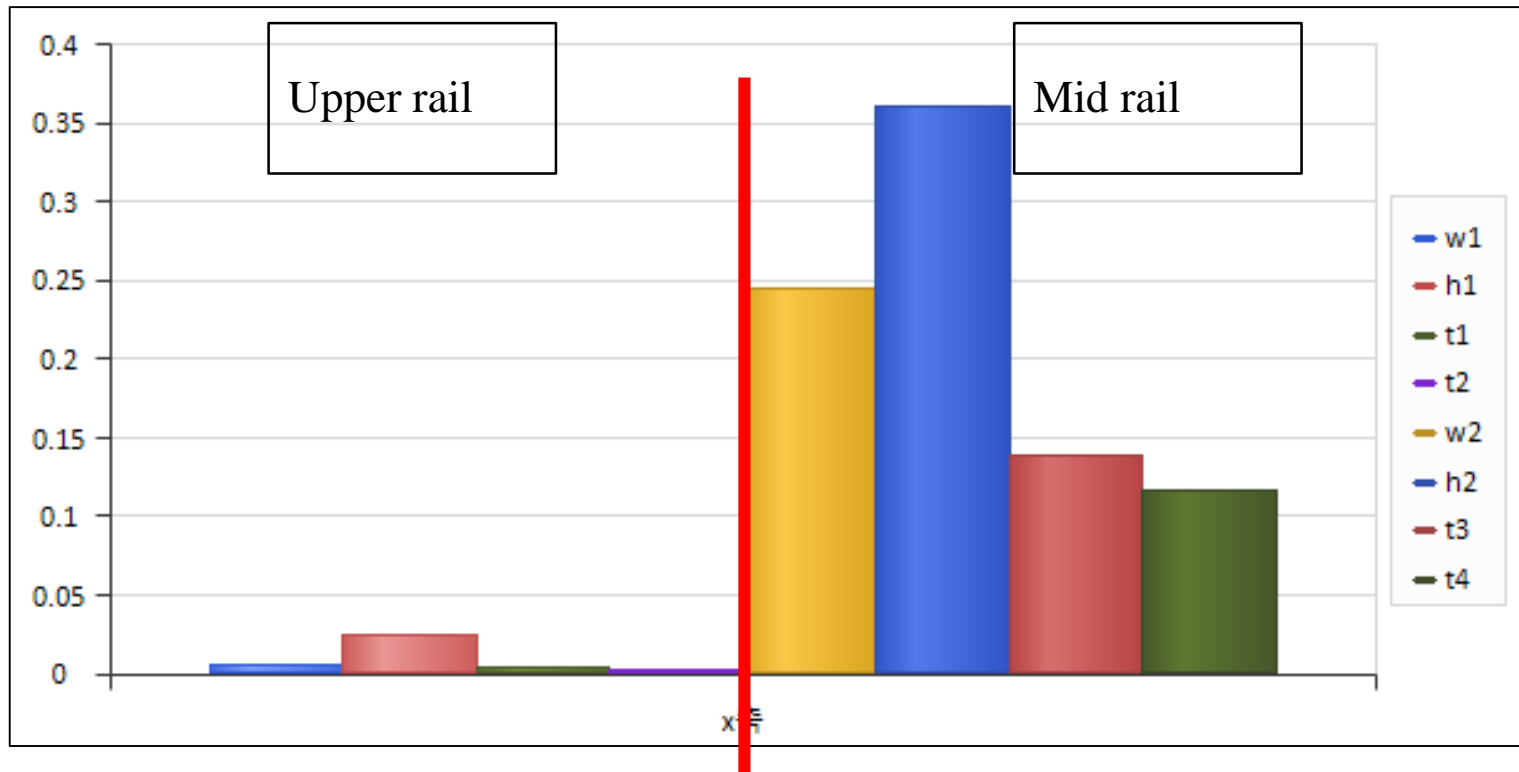
- Minimize z direction displacement

Design constraints

번호	이름	최소값	초기값	최대값
1	w1	5.000000	50.000000	95.000000
2	h1	5.000000	50.000000	95.000000
3	t1	0.100000	1.000000	1.900000
4	t2	0.100000	1.000000	1.900000
5	w2	5.000000	50.000000	95.000000
6	h2	5.000000	50.000000	95.000000
7	t3	0.100000	1.000000	1.900000
8	t4	0.100000	1.000000	1.900000

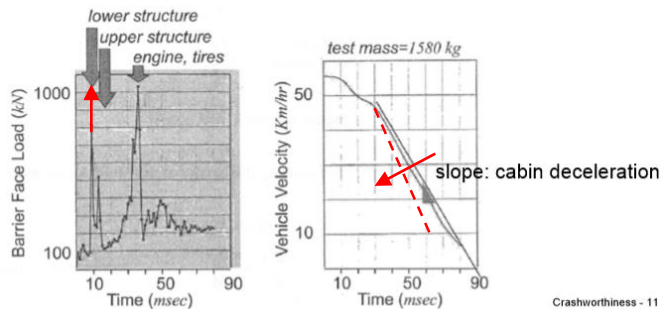
Fixed volume

Correlation analysis



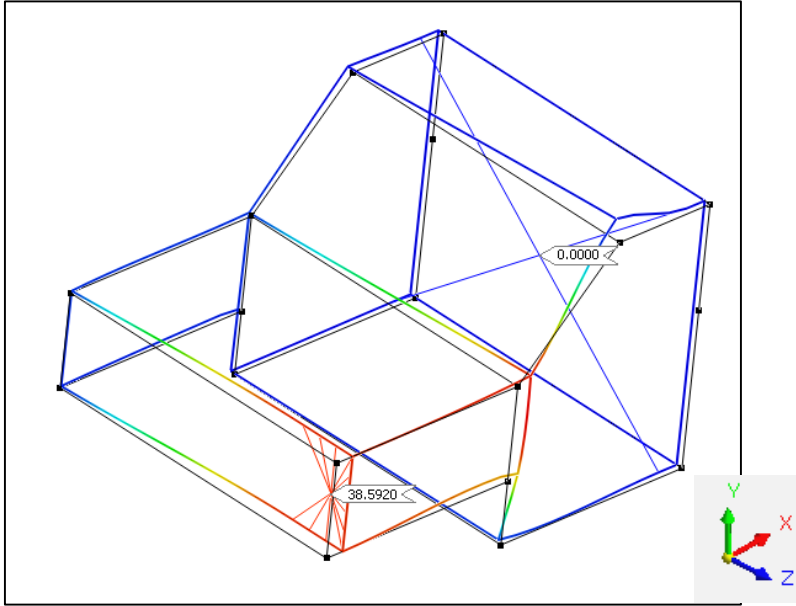
Size optimization

설계변수 이름	초기값	최소값	최대값	설계안 1	설계안 2	설계안 3	사용자 설계안
				입력			
w1	50	5	95	34	34	31	34
h1	50	5	95	20	23	24	20
t1	1	0.1	1.9	0.69	0.56	0.57	0.69
t2	1	0.1	1.9	1	0.55	0.62	1
w2	50	5	95	74	71	60	74
h2	50	5	95	86	69	80	86
t3	1	0.1	1.9	0.99	1	0.9	0.99
t4	1	0.1	1.9	0.75	0.84	0.96	0.75
				출력 (예상값 / 해석값)			
목적함수 변화율 (%)	0			-37	-29	-29	
제약조건 최대위배율 (%)	1.6			6.1	14	13	
목적함수-1	21			13	15	15	
제약조건-1	7.9e+005	7.8e+005	7.8e+005	7.3e+005	6.7e+005	6.8e+005	
*							

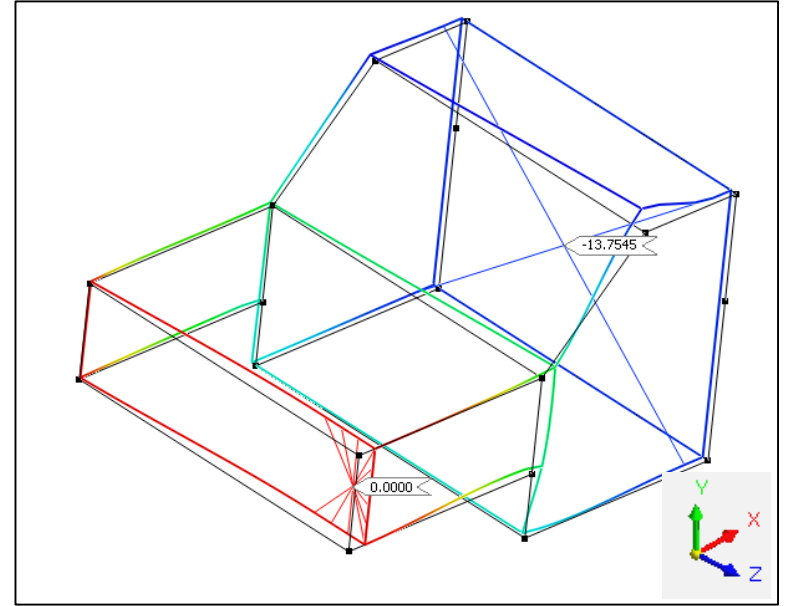


Crashworthiness - 11

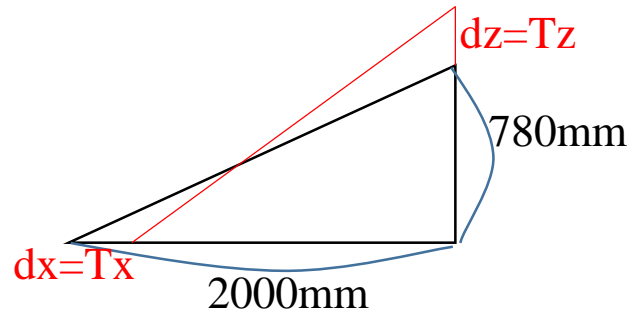
Simulation



$T_x = 38.6\text{mm}$



$T_z = 13.8\text{mm}$



$\Delta\theta = 0.98\text{deg}$

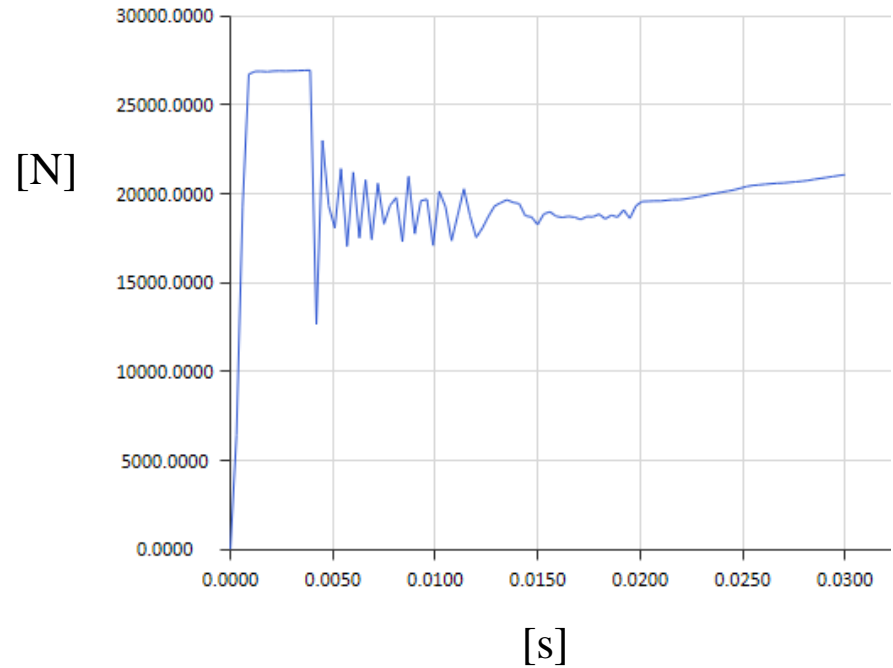
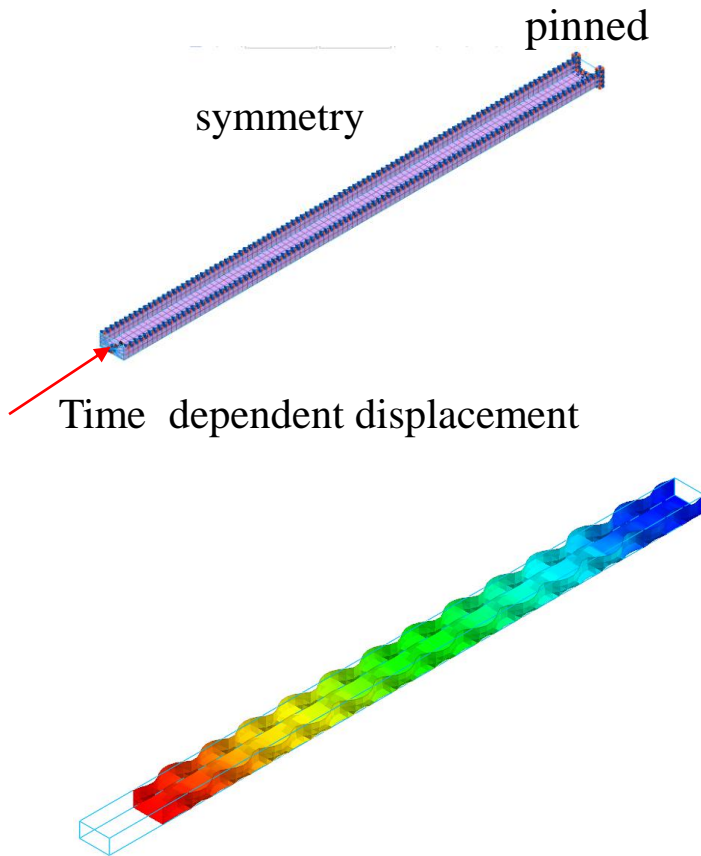


$\Delta\theta = 0.73\text{deg}$

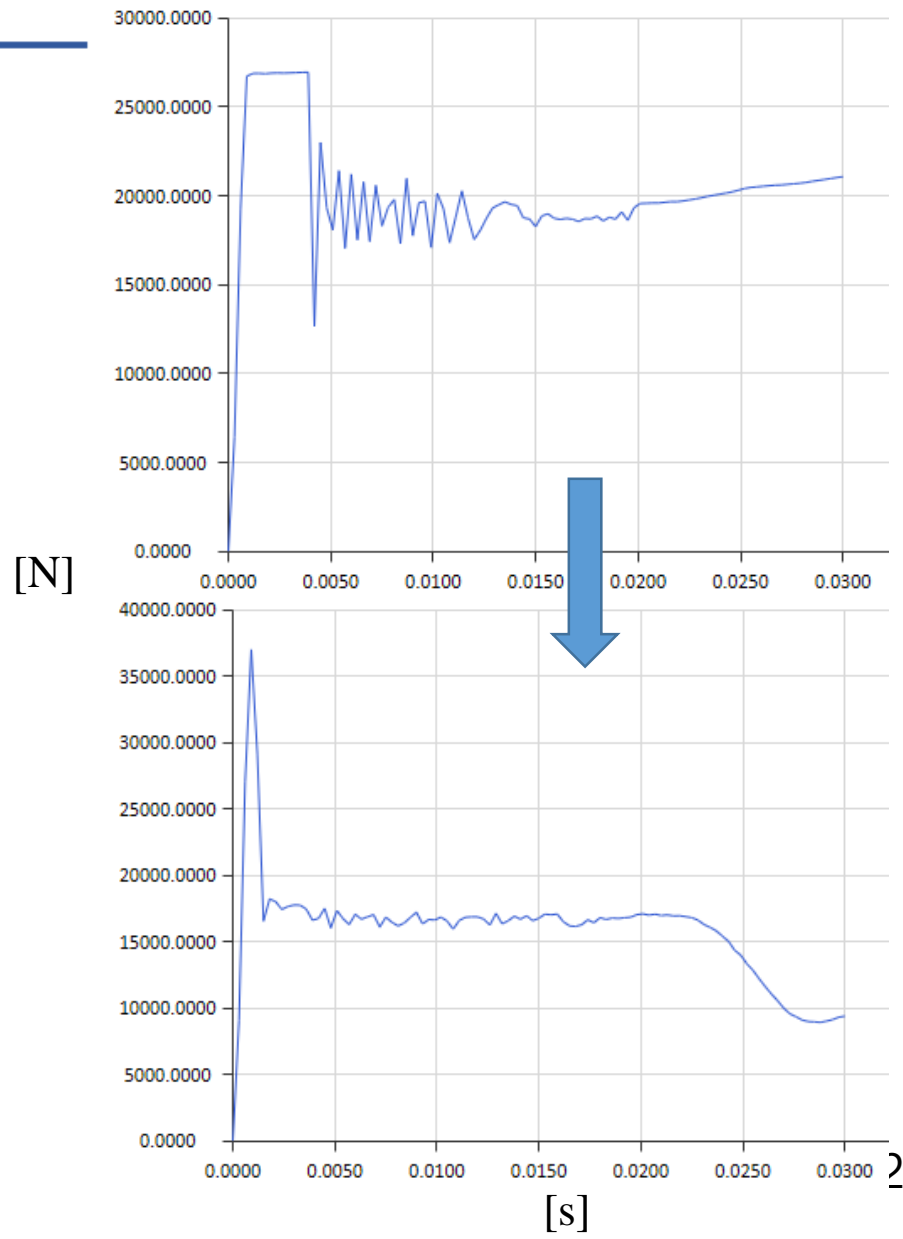
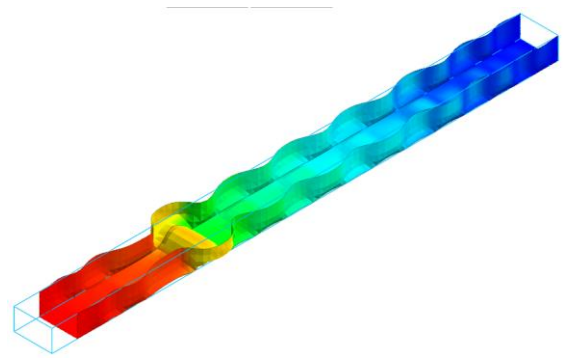
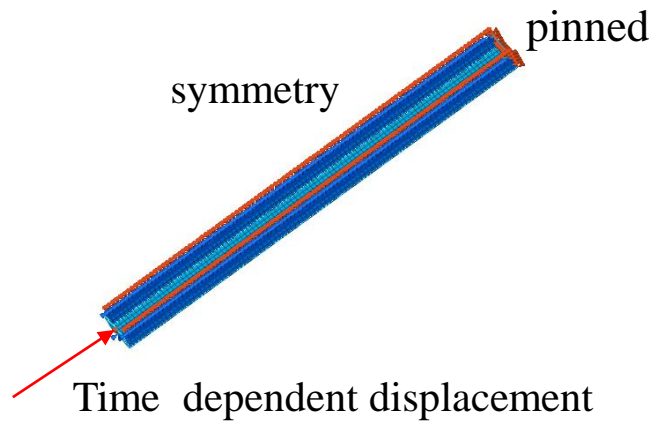
25.5% reduced

Energy absorption test

Before optimization



After optimization



Conclusion

Conclusion

- Reduce yaw motion by size optimization of motor compartment rails
- Trade off : energy absorption decrease
- In real world yaw motion is more risky for passenger
reinforce safety by small yaw motion after small overlap

Thank you!

Reference

Chen, H. & Yang, Y. & Wang, L.. (2015). Vehicle front structure energy absorbing optimization in frontal impact. 9. 168-172. 10.2174/1874155X01509010168.